

FREE VIBRATION ANALYSIS OF WALL-FRAME STRUCTURES BY DIFFERENTIAL QUADRATURE METHOD

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Abstract- Differential Quadrature Method (DQM) has very wide applications in the field of structural vibration. The main advantages of the Differential Quadrature Method are its inherent conceptual simplicity and the fact that easily programmable. In this paper free vibration analysis of wall-frame structures were studied. A wall-frame structure was modeled as an cantilever beam in this study. The governing differential equation of wall-frame structures were solved using Differential Quadrature Method (DQM). At the end of the study, a sample taken from literature was solved and the results were evaluated in order to test the convenience of the method.

Key Words- Free Vibration, Wall-Frame Structures, Differential Quadrature Method.

1.INTRODUCTION

A number of methods, such as finite element method, has been developed for analyses of buildings. The continuum model is a very simple and efficient method used in static and dynamic analysis of shear wall-frame buildings. There are numerous studies [1-44] in the literature regarding the continuum method. In continuum model a wall-frame structure was modeled as an cantilever beam. The governing differential equations of equivalent shear flexure cantilever beam are formulated using the continuum approach.

The Differential Quadrature Method (DQM) was initially presented by Bellman et al. [45-46] as an efficiently and accurate numerical method to solve differential equations. Afterwards many researchers demonstrated their successful applications of the method in mechanics [47-64]. In this study free vibration analysis of wall-frame structures were studied using Differential Quadrature Method (DQM). The following assumptions are made in this study; the behavior of the material is linear elastic, small displacement theory is valid, P-delta effects and axial deformation in columns and walls are negligible, the structures are regular (i.e. their characteristics do not vary over the height), the floor slabs of the buildings have great in-plane and small out-of- plane stiffness, torsion effects are negligible.

2.ANALYSIS

2.1. Physical Model

The behavior of the wall-frame structures ignoring axial deformations of wall and columns may be presented by combination of flexural cantilever beam and shear cantilever beam deforming in bending and shear configurations (Fig. 1.) [22-42].



Figure 1. Physical model of equivalent sandwich beam

In the Figure 1. EI are the total bending rigidities of shear walls, (K_s) are the equivalent shear rigidity of the storey for framework. For frame elements which consists of n columns and n-1 beams, Ks can be calculated as follows [7, 23]:

$$K_{s} = \frac{12E}{h_{i}(1/\sum_{1}^{n} I_{c}/h + 1/\sum_{1}^{n-1} I_{g}/l)}$$
(1)

where $\sum I_c / h$ represents the sum of moments of inertia of the columns per unit height of frame, and $\sum I_g / l$ represents the sum of moments of inertia of each beam per unit span across one floor of frame.

2.2. Exact Solution of Governing Equation of Wall-Frame Structures

The governing equation for free vibration of wall-frame structures can be written as [22-42].

$$\frac{\rho}{EI}\frac{\partial^2 u(\xi,t)}{\partial t^2} + \frac{1}{H^4}\frac{\partial^4 u(\xi,t)}{\partial \xi^4} - H^2\frac{K_s}{EI}\frac{\partial^2 u(\xi,t)}{\partial \xi^2} = 0$$
(2)

where ρ is the mass per unit length in the model, H is the total height of the building $u(\xi, t)$ is the lateral displacements at non-dimensional height $\xi=z/H$ (varying between zero at the base of the building and one at roof level) at time t.

If a sinusoidal variation of u with circular frequency ω is assumed then

$$u(\xi, t) = y(\xi)\sin(\omega t)$$
(3)

Where $y(\xi)$ is the amplitude of the sinusoidally varying displacement.

Substituting Eq.(3) in Eq. (2) results

$$\frac{d^4y}{dz^4} - k^2 \frac{d^2y}{dz^2} - \alpha y = 0$$
(4)

in which $k = H \sqrt{\frac{K_s}{EI}}$ and $\alpha = \frac{\rho}{EI} H^4 \omega^2$

The boundary conditions of a problem are;

 $y(0) = 0 \tag{5a}$

$$\frac{dy(0)}{d\xi} = 0 \tag{5b}$$

$$\frac{EI}{H^2} \frac{d^2 y(1)}{d\xi^2} = 0$$
(5c)

$$\frac{d^3 y(1)}{d\xi^3} - k^2 \frac{dy(1)}{d\xi} = 0$$
(5d)

Under the boundary conditions the exact solution of Equation (4), the circular frequencies are obtained as

$$\omega = \frac{2\pi\eta}{H^2} \sqrt{\frac{EI}{\rho}}$$
(6)

For the first three modes values for frequency parameter (η) are given in Table 1, Table 2 and Table 3 as a function of parameter k [41].

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k	Н	k	η	k	η	k	η	k	η
0	0.5596	4.5	1.465	9.5	2.680	14.5	3.913	20	5.278
0.1	0.5606	5.0	1.586	10	2.803	15.0	4.036	30	7.769
0.5	0.5851	5.5	1.706	10.5	2.926	15.5	4.160	40	10.26
1.00	0.6542	6.0	1.827	11.0	3.049	16.0	4.284	50	12.76
1.50	0.7511	6.5	1.949	11.5	3.172	16.5	4.408	60	15.26
2.00	0.8628	7.0	2.070	12.0	3.295	17.0	4.532	70	17.76
2.50	0.9809	7.5	2.192	12.5	3.418	17.5	4.656	80	20.26
3.00	1.1014	8.0	2.313	13.0	3.542	18.0	4.781	90	22.76
3.5	1.2226	8.5	2.435	13.5	3.665	18.5	4.905	100	25.26
4.00	1.3437	9.0	2.558	14.0	3.789	19.0	5.029	>10000	k/4

Table 1 Frequency parameter (η) for the first natural frequency

Table 2 Frequency parameter (η) for the second natural frequency

k	η	k	η	k	η	k	η	k	η
0	3.507	4.5	5.290	9.5	8.643	14.5	12.19	20	16.18
0.1	3.508	5.0	5.606	10	8.992	15.0	12.55	30	23.54
0.5	3.536	5.5	5.929	10.5	9.342	15.5	12.91	40	30.97
1.00	3.622	6.0	6.257	11.0	9.694	16.0	13.27	50	38.43
1.50	3.760	6.5	6.590	11.5	10.05	16.5	13.63	60	45.90
2.00	3.943	7.0	6.926	12.0	10.40	17.0	13.99	70	53.38
2.50	4.165	7.5	7.266	12.5	10.76	17.5	14.35	80	60.86
3.00	4.416	8.0	7.607	13.0	11.11	18.0	14.72	90	68.35
3.5	4.691	8.5	7.951	13.5	11.47	18.5	15.08	100	75.840
4.00	4.984	9.0	8.296	14.0	11.83	19.0	15.45	>10000	3k/4

k	η	k	Н	k	η	k	η	k	η
0	9.819	4.5	11.65	9.5	16.27	14.5	21.72	20	28.06
0.1	9.820	5.0	12.04	10	16.79	15.0	22.28	30	40.01
0.5	9.844	5.5	12.44	10.5	17.32	15.5	22.85	40	52.21
1.00	9.919	6.0	12.87	11.0	17.85	16.0	23.42	50	64.53
1.50	10.04	6.5	13.32	11.5	18.39	16.5	23.99	60	76.90
2.00	10.21	7.0	13.78	12.0	18.94	17.0	24.57	70	89.31
2.50	10.42	7.5	14.26	12.5	19.49	17.5	25.15	80	101.74
3.00	10.68	8.0	14.75	13.0	20.04	18.0	25.72	90	114.19
3.5	10.97	8.5	15.25	13.5	20.59	18.5	26.30	100	126.65
4.00	11.30	9.0	15.75	14.0	21.15	19.0	26.89	>10000	5k/4

Table 3 Frequency parameter (η) for the third natural frequency

2.3. Solution Governing Equation of Wall-Frame Structures via Differential Quadrature Method (DQM)

As shown in Figure 2 we consider a one- dimensional problem. It is assumed that a function $y(\xi)$ is smooth over the whole domain.



Figure 2 Discretization of one Dimensional Problem

If it assumed that y is a polynomial of degree n-1, y function can be written as;

$$y = \sum_{i=1}^{n} a_{i-1} * \xi^{i-1}$$
(7)

where a_{i-1} are the coefficients, n is the number of nodes.

Using Equation (7), the first second, third and fourth derivatives of y function respect to ξ can be written as

$$\frac{dy}{d\xi} = \sum_{i=1}^{n} (i-1) * a_{i-1} * \xi^{i-2}$$
(8)

$$\frac{d^2 y}{d\xi^2} = \sum_{i=1}^n (i-1) * (i-2) * a_{i-1} * \xi^{i-3}$$
(9)

$$\frac{d^3 y}{d\xi^3} = \sum_{i=1}^n (i-1)^* (i-2)^* (i-3)^* a_{i-1}^* \xi^{i-4}$$
(10)

$$\frac{d^4y}{d\xi^4} = \sum_{i=1}^n (i-1) * (i-2) * (i-3) * (i-4) * a_{i-1} * \xi^{i-5}$$
(11)

Functional values in the whole domain can be written as;

. .

$$y_1 = y(0) = a_0 \tag{12.1}$$

$$y_1 = y(h) = a_0 + a_1 h$$
 (12.2)

$$y_{n} = y(n*h-h) = a_{0} + a_{1}*(h*n-h) + \dots + a_{n}*(h*n-h)^{n}$$
(12.n)

Where h is the distance of two grid points and is equal to 1/(n-1).

Equation (13) shows the matrix form of equations (12.1),(12.2)..... (12.n)

$$\begin{bmatrix} y_{1} \\ y_{2} \\ \vdots \\ \vdots \\ y_{n-1} \\ y_{n} \end{bmatrix} = \begin{bmatrix} 1 & 0 & \dots & 0 & 0 \\ 1 & h & \dots & h^{n} & 0 \\ \vdots & \vdots & \ddots & \ddots & h^{n} & 0 \\ \vdots & \vdots & \ddots & \ddots & \ddots & 0 \\ \vdots & \vdots & \vdots & \ddots & \ddots & 0 \\ \vdots & \vdots & \vdots & \vdots & \vdots \\ 1 & (n-2)^{*}h & \vdots & \vdots & (n-2)^{n} * h^{n} \\ 1 & (n-1)^{*}h & \vdots & \vdots & (n-1)^{n} * h^{n} \end{bmatrix} \begin{bmatrix} a_{0} \\ a_{1} \\ \vdots \\ \vdots \\ a_{n-1} \\ a_{n} \end{bmatrix} = A \begin{bmatrix} a_{0} \\ a_{1} \\ \vdots \\ \vdots \\ a_{n-1} \\ a_{n} \end{bmatrix}$$
(13)

Using Equations (11), (9) and (7) Equation (4) can be written as

$$\sum_{i=1}^{n} (i-1)*(i-2)*(i-3)*(i-4)*a_{i-1}*\xi^{i-5} - \sum_{i=1}^{n} (i-1)*(i-2)*a_{i-1}*\xi^{i-3} - \sum_{i=1}^{n} a_{i-1}*\xi^{i-1} = 0$$
(14)

When consider the boundary conditions Equations (5a), (5b),(5c) and (5d) can be written as

$$y_1 = y(0) = a_0 = 0 \tag{15}$$

$$\frac{dy(0)}{d\xi} = a_1 = 0 \tag{16}$$

$$\frac{d^2 y(1)}{d\xi^2} = \sum_{i=1}^n (i-1)^* (i-2)^* a_{i-1}^{-1} * 1^{i-3} = 0$$
(17)

$$\frac{d^{3}y(1)}{d\xi^{3}} - k^{2}\frac{dy(1)}{d\xi} = \sum_{i=1}^{n} (i-1)*(i-2)*(i-3)*a_{i-1}*1^{i-4} - k^{2}\sum_{i=1}^{n} (i-1)*(i-2)*a_{i-1}*1^{i-3} = 0$$
(18)

Using Equations (14),(15),(16),(17) and (18) the matrix equation (19) can be written

$$B\begin{bmatrix} a_{0} \\ a_{1} \\ \cdot \\ \cdot \\ a_{n-1} \\ a_{n} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ \cdot \\ \cdot \\ \cdot \\ 0 \\ 0 \end{bmatrix}$$
(19)

The elements of B matrix can be written as follows;

B(1,1) = 1 (20)

B(1,i) = 0 i = 2....n (21)

$$B(2,1) = 0 (22)$$

$$B(2,2) = 1$$
 (23)

$$B(2,i) = 0 i = 3....n (24)$$

$$B(n-1,i) = (i-1)^*(i-2)^*1 \qquad i=1...n$$
(25)

$$B(n,i) = (i-1)*(i-2)*(i-3) \qquad i = 1...n$$
(26)

$$B(j,i) = (i-1)^{*}(i-2)^{*}(i-3)^{*}(i-4)^{*}(j^{*}h-h)^{i-4} - k^{2}(i-1)^{*}(i-2)^{*}(j^{*}h-h)^{i-2} - \alpha(j^{*}h-h)^{i} \quad j = 3...n-2$$
(27)

When coefficients vector is solved out from Equation (19) and is substituted to the Equation (13), Equation (28) is obtained.

[0]]	$\begin{bmatrix} y_1 \end{bmatrix}$]	$\begin{bmatrix} y_1 \end{bmatrix}$
0		<i>y</i> ₂		<i>y</i> ₂
.				•
	$=BA^{-1}$		=C	
0		y_{n-1}		y_{n-1}
0		y_n		y_n

The values of ω which set the determinant of C matrix to zero are the circular frequencies.

3. A VALIDITY OF THE METHOD

In this part of the study for a different values of k parameter, is obtained frequency parameter (η) using Differential Quadrature method and compared with the exact values in Table 4, Table 5 and Table 6. A program for Differential Quadrature Method was prepared in MATLAB.

Table 4 Comparison of Frequency parameter (η) for the first natural frequency

k	Analytical	DQM	DQM	DQM	
		(n=8)	(n=10)	(n=12)	
0	0.5596	0.5598	0.5596	0.5596	
2	0.8628	0.8630	0.8627	0.8628	
5	1.5860	1.5836	1.5852	1.5856	
10	2.803	2.7940	2.7904	2.7992	
15	4.036	4.0721	4.0119	4.0151	
20	5.278	5.4038	5.2769	5.2418	
50	12.76	13.5890	13.2292	13.0107	
100	25.26	27.2447	26.5671	26.1624	

k	Analytical	DQM (n=8)	DQM (n=10)	DQM (n=12)
0	3.5070	3.5610	3.5062	3.5070
2	3.9430	4.0228	3.9430	3.9434
5	5.6060	5.8078	5.6175	5.6073
10	8.9920	9.5460	9.1260	9.0219
15	12.5500	13.5613	12.9505	12.6929
20	16.1800	17.6965	16.9295	16.5389
50	38.4300	43.1441	41.4873	40.5542
100	75.8400	85.9639	82.7547	80.9567

Table 5 Comparison of frequency parameter (η) for the second natural frequency

Table 6 Comparison of frequency parameter (η) for the third natural frequency

k	Analytical	DQM	DQM (n=10)	DQM (n=12)
0	9.8190	8.327	10.3306	9.7589
2	10.2100	8.715	10.7197	10.1496
5	12.04	10.642	12.5208	11.9811
10	16.79	16.1474	17.2422	16.7856
15	22.28	22.6529	22.9011	22.3987
20	28.06	29.3597	29.0266	28.3935
50	64.53	70.3910	68.5277	67.1584
100	126.65	139.7029	135.9768	133.4607

4. CONCLUSIONS

In this paper free vibration analysis of wall-frame structures were studied. A wall-frame structure was modeled as an cantilever beam in this study. The governing differential equation of wall-frame structures were solved using Differential Quadrature Method (DQM). At the end of the study, it was observed from the literature that the presented method gave results sufficient. For n=12, in the first mode the error of the DQM is shown to be less than 4%. Because of the main advantages of the Differential Quadrature Method are its inherent conceptual simplicity and the fact that easily programmable it can be used at the concept design stage.

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