

ISSN 0570 - 1414

# COMMUNICATIONS

DE LA FACULTÉ DES SCIENCES  
DE L'UNIVERSITÉ D'ANKARA

Série B : Chimie

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TOME : 31

ANNÉE : 1985

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by

Mustafa ALPBAZ      Mete KOÇKAR

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Faculté des Sciences de l'Université d'Ankara  
Ankara, Turquie

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TURQUIE

## Stability Analysis For Well Stirred Tank Cooled By Jacketed

Mustafa ALPBAZ and Mete KOÇKAR

(Received November 2, 1984 and accepted March 6, 1985)

Chemical Engineering Department, Faculty of Science,  
Ankara University - ANKARA

Chemical Engineering Department, Engineering Faculty  
Anadolu University - ESKİŞEHİR

### SUMMARY

In this work, Routh stability analysis was applied to choose the suitable control parameters for the feedback control of the well stirred tank cooled by jacketed. For this purposes the related mathematical models (1) was converted to the Laplace Transform and the values of parameters were calculated with stability test.

### INTRODUCTION

A stable system can be defined as one for which the output response is bounded for all bounded inputs. A system exhibiting an unbounded response to a bounded input is unstable.

The stability criterion can be used to ascertain the stability of control system of the form shown in Fig. 1. From the block diagram of the control system, the related equation can be obtained.

$$C = \frac{G_1 G_2}{1 + G_1 G_2 H} R + \frac{G_2}{1 + G_1 G_2 H} U \quad (1)$$

$$G = G_1 G_2 H \quad (2)$$

We call,  $G$ , the open-loop transfer functions. A linear control system is unstable, if any roots of its characteristic equation,  $1 + G_1 G_2 H$ , are on the imaginary axis. Otherwise the system is stable. The Routh test for stability is a purely algebraic method for determining how many roots of the characteristic equation have positive real parts; from this

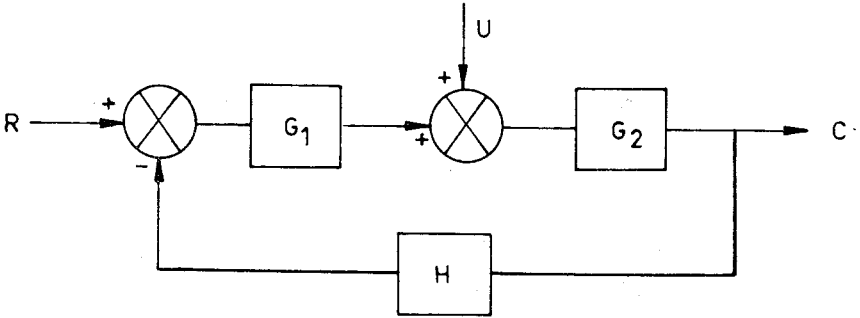


Figure. 1. Simple feedback control system.

it can also be determined if the system is stable, for if there are no roots with positive real parts, the system is stable. The test is limited to system which have polynomial characteristic equations. This stability criterion can be found in many text book (2)

In present work, Routh stability test were applied to the well mixed tank cooled by jacked.

### MATHEMATICAL MODELS AND STABILITY ANALYSIS

The related mathematical models dealing with well mixed tank cooled by jacked were driven in the work of Alpaz and Erdoğan (3,4). The linearized models are shown below;

$$\frac{dT'_{po}}{dt} = \left( \frac{T^o_{pi} - T^o_{po}}{M_v} \right) M'_p - \left( \frac{UA + M_p C_p}{M_v C_p} \right) T'_{po} + \left( \frac{UA}{2M_v C_p} \right) T'_{co} \quad (3)$$

$$\frac{dT'_{co}}{dt} = \left( \frac{T^o_{ci} - T^o_{co}}{M_j C_c} \right) M'_c - \left( \frac{M_c C_c + \frac{UA}{2}}{M_j C_c} \right) T'_{co} + \left( \frac{UA}{M_j C_c} \right) T'_{po} \quad (4)$$

The range of parameters and the solution of related models with Laplace transform and digital computer solutions are found in the work of Alpaz, Erdoğan and Koçkar (5).

Consider the system differantial equations (3,4) with numerical values, Table. 1.

$$\frac{dT'_{po}}{dt} = \left( \frac{64-71.49}{32595} \right) M'_p - \left( \frac{6+3.45015}{21512.7} \right) T'_{po} + \left( \frac{6}{(2)(21512.7)} \right) T'_{co} \quad (5)$$

$$\frac{dT'_{co}}{dt} = \left( \frac{17-33.21}{4922.8} \right) M'_c - \left( \frac{17 + \frac{6}{2}}{4922.8} \right) T'_{co} + \left( \frac{6}{4922.8} \right) T'_{po} \quad (6)$$

$$\frac{dT'_{po}}{dt} = - (2.2978 \times 10^{-4}) M'_p - (4.3928 \times 10^{-4}) T'_{po} + (1.394 \times 10^{-4}) T'_{co} \quad (7)$$

$$\frac{dT'_{co}}{dt} = - (4.062 \times 10^{-3}) T'_{co} + (1.218 \times 10^{-3}) T'_{po} - (3.292 \times 10^{-3}) M'_c \quad (8)$$

Transforming;

$$\bar{T}'_{po} = - \left( \frac{2.2978 \times 10^{-4}}{s + 4.3928 \times 10^{-4}} \right) \bar{M}'_p + \left( \frac{1.394 \times 10^{-4}}{s + 4.3928 \times 10^{-4}} \right) \bar{T}'_{co} \quad (9)$$

$$\bar{T}'_{co} = \left( \frac{1.218 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) \bar{T}'_{po} - \left( \frac{3.292 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) \bar{M}'_c \quad (10)$$

The stability analysis were applied for each control system.

i-Proportional Control

$$\bar{M}'_c = - K_c \bar{T}'_{po} \quad (11)$$

If equation (11) is put into equation (10);

$$\bar{T}'_{co} = \left( \frac{1.218 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) \bar{T}'_{po} - \left( \frac{3.292 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) (-K_c \bar{T}'_{po}) \quad (12)$$

$$\bar{T}'_{co} = \left( \frac{3.292 \times 10^{-3} K_c + 1.218 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) \bar{T}'_{po} \quad (13)$$

If equation (13) is put into equation (9);

$$T'_{po} = - \left( \frac{2.2978 \times 10^{-4}}{s + 4.392 \times 10^{-4}} \right) \bar{M}'_p + \left( \frac{1.394 \times 10^{-4}}{s + 4.392 \times 10^{-4}} \right) \quad (14)$$

$$\left( \frac{3.292 \times 10^{-3} K_c + 1.218 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) \bar{T}'_{po}$$

and then

$$\bar{T}'_{po} = -(2.297 \times 10^{-4})$$

$$\left[ \frac{s + 4.062 \times 10^{-3}}{s^2 + 4.5012 \times 10^{-3} s + 1.6143 \times 10^{-6} - 4.5892 \times 10^{-7} K_c} \right] \bar{M}'_p \quad (15)$$

The characteristic equations;

$$s^2 + 4.5012 \times 10^{-3} s + (1.6143 \times 10^{-6} - 4.5892 \times 10^{-7} K_c) = 0 \quad (16)$$

For stable time response, the roots should be real;

$$(4.5012 \times 10^{-3})^2 - 4(1.6143 \times 10^{-6} - 4.5892 \times 10^{-7} K_c) > 0 \quad (17)$$

$$1.3803 \times 10^{-5} + 1.8356 \times 10^{-6} K_c > 0 \quad (18)$$

Therefore,

$$K_c > -7.5$$

For Routh stability test, if any coefficient of the characteristic equation is negative or zero a system is unstable,

1	1	$1.6143 \times 10^{-6} - 4.5892 \times 10^{-7} K_c$
2	$4.5012 \times 10^{-3}$	
3	$1.6143 \times 10^{-6} - 4.5892 \times 10^{-7} K_c$	

Hence,

$$K_c < 3.51$$

ii- Proportional + Integral control

$$\bar{M}'_c = -K_c \left( 1 + \frac{K_i}{K_c} \frac{1}{s} \right) \bar{T}'_{po} \quad (19)$$

If equation (19) is put into equation (10),

$$\bar{T}'_{co} = \left( \frac{1.218 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) \bar{T}'_{po} - \left( \frac{3.292 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right)$$

$$\left[ -K_c \left( 1 + \frac{K_i}{K_c} \frac{1}{s} \right) \right] \bar{T}'_{po} \quad (20)$$

$$\bar{T}'_{co} = \left[ \frac{(1.218 \times 10^{-3} + 3.292 \times 10^{-3} K_c) s + 3.292 \times 10^{-3} K_i}{s(s + 4.062 \times 10^{-3})} \right] \bar{T}'_{po} \quad (21)$$

Put this equation (21) into equation (9).

$$\bar{T}'_{po} = - \left( \frac{2.297 \times 10^{-4}}{s + 4.392 \times 10^{-4}} \right) \bar{M}'_p + \left( \frac{1.392 \times 10^{-4}}{s + 4.392 \times 10^{-4}} \right) \left[ \frac{(1.218 \times 10^{-3} + 3.292 \times 10^{-3} K_c) s + 3.292 \times 10^{-3} K_i}{s (s + 4.062 \times 10^{-3})} \right] \bar{T}'_{po} \quad (22)$$

The characteristic equation,

$$s^3 + 4.5012 \times 10^{-3} s^2 + (1.614 \times 10^{-6} - 4.589 \times 10^{-7} K_c) s - 4.589 \times 10^{-7} K_i = 0 \quad (23)$$

If Routh test is applied,

For stability,  $(-4.589 \times 10^{-7} K_i)$  and  $(1.614 \times 10^{-6} - 4.589 \times 10^{-7} K_c + 1.01 \times 10^{-4} K_i)$  must be positive. In this case  $K_i < 0.0$  and

$$T_R = \frac{K_c}{K_i} \text{ being positive and than } K_c < 0.0$$

iii- Proportional + Derivative + Integral Control

$$\bar{M}'_c = - K_c \left( 1 + \frac{K_d}{K_c} s + \frac{K_i}{K_c} \frac{1}{s} \right) \bar{T}'_{po} \quad (24)$$

Put into equation (10),

$$\bar{T}'_{co} = \left( \frac{1.218 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) \bar{T}'_{po} - \left( \frac{3.292 \times 10^{-3}}{s + 4.062 \times 10^{-3}} \right) \quad (25)$$

$$\left[ - K_c \left( 1 + \frac{K_d}{K_c} s + \frac{K_i}{K_c} \frac{1}{s} \right) \right] \bar{T}'_{po} \quad (25)$$

$$\bar{T}'_{co} = \left[ \frac{(1.218 \times 10^{-3} + 3.292 \times 10^{-3} K_c) s + 3.292 \times 10^{-3} K_d s^2 + 3.292 \times 10^{-3} K_i}{s (s + 4.062 \times 10^{-3})} \right] \quad (26)$$

If equatin (26) is put into equation (9), characteristic equation is become,

$$s^3 + (4.501 \times 10^{-3} - 4.587 \times 10^{-7} K_d) s^2 + (1.614 \times 10^{-6} - 4.587 \times 10^{-7} K_c) s - 4.587 \times 10^{-7} K_i = 0 \quad (27)$$

Hence  $(-4.587 \times 10^{-7} K_i)$  requires that  $K_i < 0.0$  and than  $K_c < 0.0$

$$\text{and } T_D = \frac{K_D}{K_c} \text{ should be positive and } K_d < 0.0$$

## RESULTS

In the previous work (5), the time response of the tank output temperature was calculated and it was observed the variation of output temperature with time was stable when tank was under the effect of the step change given to the feed flow rate.

In the present work, the feedback control system was added to force the output temperature for reaching to the set point. In control solution, three term control mechanism was introduced to the differential equations (3,4) which describe the dynamic response of the mixing tank and this equations were solved with Laplace transform. When the system was in the steady-state having operating condition shown in Table. 1, the step change was given to the feed flow rate and it was investigated the effect of the control parameters on the output temperature. The effect of the proportional acting factor,  $K_c$ , on the output

Table.1. The operating conditions and control parameters for feedback control of the tank

$M_p \left( \frac{g}{sec} \right)$	$M_c \left( \frac{g}{sec} \right)$	$T_{pi} (C^\circ)$	$T_{ci} (C^\circ)$	$UA \left( \frac{Cal}{sec C^\circ} \right)$	$K_c$	$K_i$	$K_d$
8.20	17.0	64.0	17.0	6.00	-5	—	—
					-5	-0.02	—
5.22	17.0	64.0	17.0	6.00	-5	-0.02	-0.02

temperature was shown in Fig. 2. When  $K_c$  was increasing the output temperature came closer to the setpoint with oscillation. But including two other control parameters,  $T_R$ , and,  $T_D$ , the time response of the output temperature become unstable, Fig. 3. For this reason Routh stability test was applied to obtain stability on the output variables and than proper control parameters were calculated as it was shown in this work. In Fig. 4. the time response of the output temperature with suitable control parameters was shown. It can be seen that the output variables are stable.



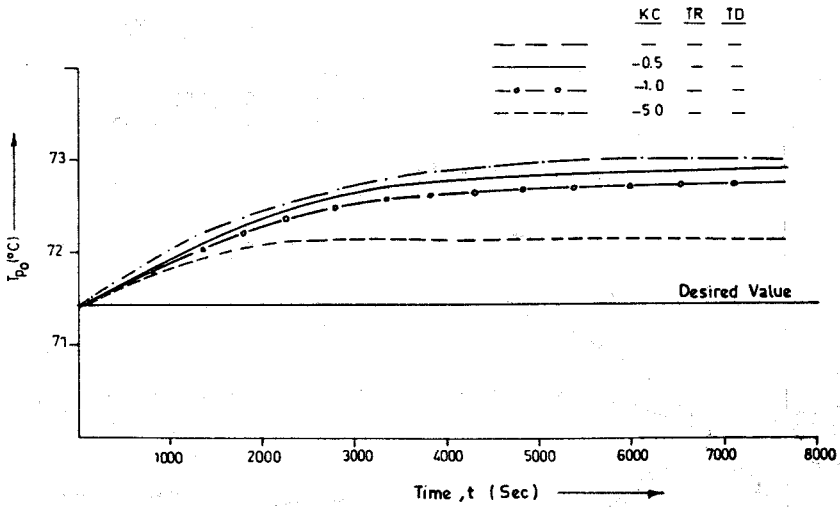


Figure. 2. The effect of the proportional acting factor  $K_c$  on the output temperature. ( $M_p^o = 8.20$  g/sec,  $M_p = 5.22$  g/sec)

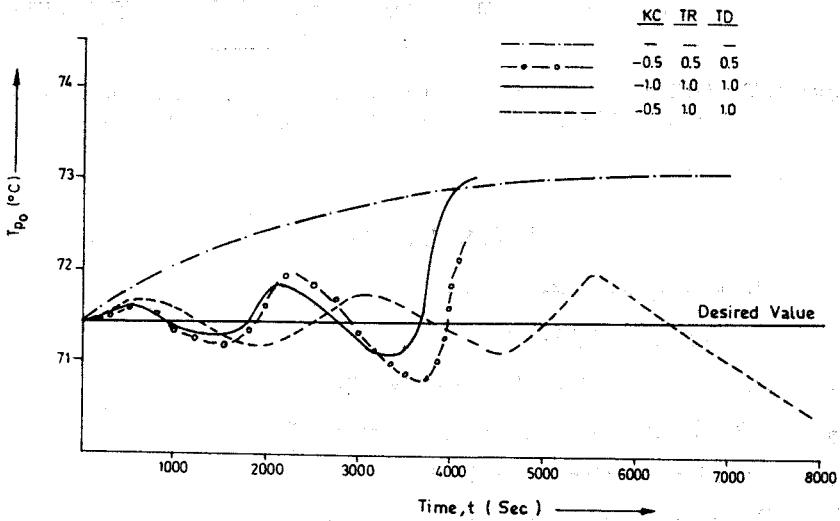


Figure. 3. The effect of the control parameters on the output tank temperature and instability ( $M_p^o = 8.20$  g/sec,  $M_p = 5.22$  g/sec)

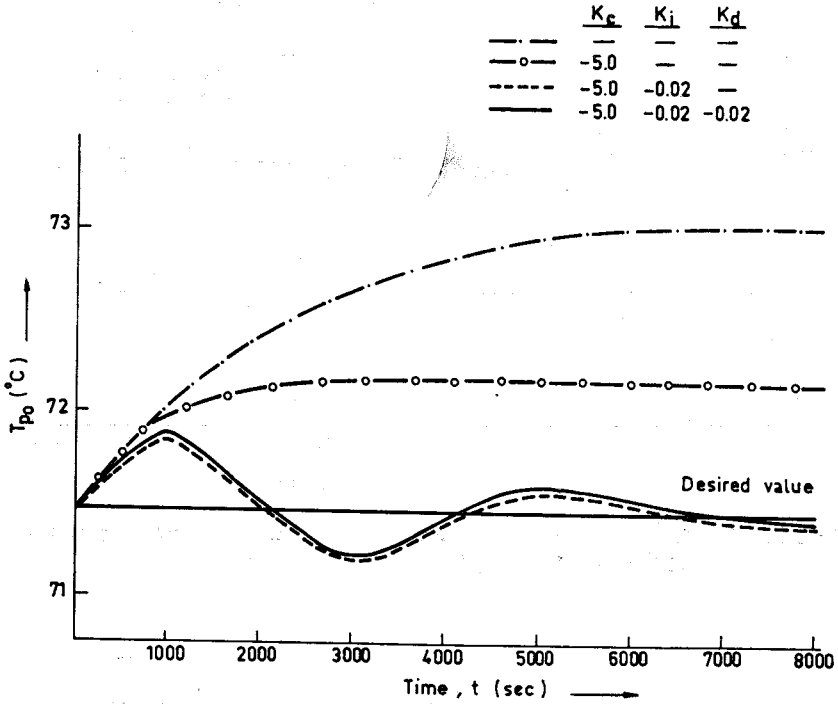


Figure. 4. The feedback control of the output tank temperature when the step change was given to the feed flow rate. ( $M^p = 8.20$  g/sec,  $M_p = 5.22$  g/sec)

### NOMENCLATURE

- A Heat transfer surface ( $\text{cm}^2$ )
- C Controlled variable
- $C_c$  Specific heat of coolant ( $\text{cal/g } ^\circ\text{C}$ )
- $C_p$  Specific heat of tank content ( $\text{cal/g } ^\circ\text{C}$ )
- $G_n$  Transfer functions of n'th elements
- H Transfer function of measuring element
- $K_c$  Proportionol acting factor.
- $K_d$  Derivative action factor

$K_i$	Integral acting factor
$M_c$	Coolant mass flow rate (g/sec)
$M_p$	Feed mass flow rate (g/sec)
$M_v$	Mass hold up in the tank (g)
$M_j$	Mass hold up in the jacked (g)
$R$	Set point
$s$	Laplace operature
$T_{pi}$	Feed temperature ( $^{\circ}C$ )
$T_{po}$	Output temperature ( $^{\circ}C$ )
$T_{co}$	Output coolant temperature ( $^{\circ}C$ )
$t$	Time
$U$	Overall heat transfer coefficient (cal/cm <sup>2</sup> sec $^{\circ}C$ ), Disturbance
$\delta$	Density (g/cm <sup>3</sup> )
$\mu$	Viscosity (g/cm sec)

### ÖZET

Bu çalışmada, dışardan ceketle soğutulan tam karıştırmalı akım tankının geri beslemeli kontrolü için kontrol parametrelerinin uygun değerlerinin seçiminde Routh kararlılık testi uygulanmıştır. Bu amaçla ilgili matematiksel modellerin (1) Laplace dönüşümleri alınmış ve kararlılık analizi ile parametrelerin değerleri hesaplanmıştır.

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