

## PERFORMANCE AND EMISSIONS ANALYSIS OF A DIESEL ENGINE FUELLED WITH ESTERS PRODUCED FROM PALM OLEIN/SOYBEAN OIL MIXTURE

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### Abstract

*In this experimental study, the engine performance and exhaust emissions of a diesel direct injection engine using mixed palm olein-soybean vegetable oil ethyl ester (POSEE) and methyl ester (POSME) were studied. The results of experimental studies have shown that the torque and the brake power output of the engine using biodiesels has a little lower value and the specific fuel consumption is higher compared to the engine using conventional diesel. It was also observed a decrease both at CO and HC emissions that indicated an advantage about exhaust emissions. Although methyl ester's CO<sub>2</sub> emission has decreased compared to the diesel fuel, NO and NOX emissions were higher with the biodiesels.*

**Keyword:** Biodiesel, Exhaust emissions, Engine Performance, Diesel Engine.

### 2. Introduction

The energy need of growing population on the world has been increasing speedily. However, fossil fuels, which are the most important energy source of today's world has been decreasing rapidly day by day while polluting the environment. Because of these problems, scientists have been studying on renewable alternative energy sources for a long time. The idea of using vegetable oils as alternative fuel for diesel engines belongs to Rudolph Diesel, who is the inventor of diesel engines<sup>1</sup>. However, several chemical properties of oils and fats are not appropriate for the engine.

The researches have shown that one way to improve the fuel properties of oils and fats is their transesterification. Transesterification is a chemical reaction which converts a fatty acid to an acid ester. This process provides a fuel, called biodiesel, which can be utilized in unmodified diesel engines<sup>2</sup>. There are several studies about biodiesel on the world. Even though the bigger portion of these studies contains methyl ester, there are also studies about ethyl and other esters. Antolin et al.<sup>3</sup>, tested engine performance and emission using four different fuel samples: diesel fuel; 30% sunflower methyl ester, 70% diesel fuel (by volume); 5% sunflower methyl ester, 95% diesel fuel; and 100% sunflower methyl ester. Dorado et al.<sup>2</sup>, studied the exhaust emissions of a diesel direct injection Perkins engine fueled with waste olive oil methyl ester at several steady-state operating conditions. Kalligeros et al.<sup>4</sup>, reported on biodiesel/marine diesel blends as a fuel in an indirect injection single cylinder diesel engine. They have used methyl esters produced from sunflower oil and olive oil as biodiesel..

Al-Widyan et al.<sup>5</sup>, tested several ester/diesel blends including 100% ester in addition to diesel fuel, which served as the baseline fuel. Puhan et al.<sup>6</sup>, prepared Mahua Oil Ethyl Ester by transesterification using sulfuric acid (H<sub>2</sub>SO<sub>4</sub>) as catalyst and tested in a 4-stroke direct injection

diesel engine with natural aspirated. The objective of another study was to investigate the engine performance and the road performance of biodiesel fuel originated from the used cooking oil in a Renault Megane Diesel engine in winter conditions for 7500 km road tests in urban and long distance traffic<sup>7</sup>. In another study, the effects of methyl ester biodiesel which was produced from a hazelnut soapstock/waste sunflower oil mixture was studied on the performance and emissions of a four cycle, four cylinder, turbocharged indirect injection (IDI) at both full and partial loads<sup>8</sup>.

Usta<sup>9</sup>, performed an experimental study on the performance and exhaust emissions of a turbocharged indirect injection diesel engine fuelled with tobacco seed oil methyl ester at full and partial loads. Raheman et al.<sup>10</sup>, presented the results of a study investigating the fuel properties of karanja methyl ester (KME) and its blend with diesel from 20% to 80% by volume and running a diesel engine with these fuels. Kalam et al.<sup>11</sup>, presented the experimental results carried out to evaluate the effect of anticorrosion additive in biodiesel (from palmoil) on diesel engine's performance, emissions and wear characteristics. Ramadhas et al.<sup>12</sup>, used various blends of biodiesel (from rubber seed oil) and diesel as fuel in compression ignition engines and analyzed its performance and emission characteristics. WinLee et al.'s study<sup>13</sup> covered short-term laboratory investigations of a 20% (by volume) soybean methyl ester blend.

In comparison to diesel, the higher cetane number of biodiesel results in shorter ignition delay and longer combustion duration and hence results low particulate emissions and minimum carbon deposits on injector nozzles<sup>14</sup>. In general, it has been reported by most researchers that using biodiesels don't require important modifications on engine. CO, HC and particulate emissions decrease, NO<sub>x</sub> emissions sometimes increase, sometimes decrease. Because of their chemical structure, SO<sub>x</sub> emissions don't occur<sup>15</sup>.

In this sense, the aim of this work is to carry out performance and emission test of a single cylinder, 4-stroke, diesel direct injection engine fueled with biodiesels (palm olein-soybean ethyl ester-POSEE and palm olein-soybean methyl ester-POSME) from mixed, refined edible palm olein-soybean oil and petroleum based diesel. The emission data includes carbon monoxide (CO), hydrocarbons (HC) and oxides of nitrogen (NO and NO<sub>x</sub>)

### 3. Biodiesel production and specifications

In the experiments, petroleum based diesel and mixed vegetable oil methyl and ethyl esters were used as fuels. The raw material oil was supplied ready-made to produce the esters. This oil had been produced mixing 35% refined palm olein oil and 65% refined soybean oil by an oil producer firm in Turkey.

In order to produce methyl ester, 1-liter oil, 200-ml. methanols and 3,5 gr KOH (as a catalyst) were used. Ethyl ester process is similar to the methyl ester process with a difference in the amount of reaction elements, which are more than the ones in methyl ester process. In order to produce it, 1-liter oil, 300 ml. ethanol and 10 gr. KOH were used<sup>16,17</sup>.

Both biodiesels physical and chemical properties, which have been analyzed in Middle East Technical University Petroleum Research Centre in Turkey and diesel fuel properties, are summarized in Table 1.

### 4. Experimental equipments and test procedure

The experimental setup consists of a Diesel engine, an engine test bed and a gas analyser. The schematic layout of the test system can be seen in Fig 1. A single cylinder, four stroke, direct injected diesel engine was selected for the tests. The technical specifications of the test engine are summarized in Table 2. The diesel engine has air-cooling system. The engine test bed (Cussons-P8160) consists of a control panel, measurement instruments and an electrical dynamometer that is rated for 24,1 Nm power absorption at 4750-rpm maximum operating speed. Cussons P8160 type electrical dynamometer is a direct current machine that turns with the diesel motor. The force, which is produced by the motor, can be measured with using load cells that is 0,25 m far away from the centre of the dynamometer. The control panel of the dynamometer carried out setting the experiment conditions and reading the measurement results. On the control panel, there is a button to set the motor load, a digital screen to read the motor load and the rotation value. The fuel consumption was measured with a burette (10 ml volume) and a stopwatch. And also, an oblique manometer was used to measure air consumption. The lubricating oil, fuel and ambient temperatures were measured by thermocouples and read from the digital screen of control panel.

The exhaust gas was given to atmosphere by a gas ventilation system. The exhaust gas emission measurements were measured with a Quintox Kane-May emission gas analyzer. The device was placed approximately 1 m away from the engine. The gas analyser, having electrochemical sensors, was used to measure the CO, CO<sub>2</sub>, NO, NO<sub>x</sub>, O<sub>2</sub> and HC emissions. This apparatus can also measure the exhaust gas temperature.

The parameters were obtained during full load and variable speed performance tests. The engine was operated on diesel fuel at first and then on POSME and POSEE fuels. After a standard warm-up procedure, the engine speed was increased to 3000 rpm. The tests and data collection were performed at seven different engine speeds; 3000, 2700, 2400, 2100, 1800, 1500 and 1200 rpm. At each speed, the engine was stabilised and then the measurement parameters were recorded.

## 5. Results and discussion

### Engine performance

Fig. 2 shows the engine torque values according to the engine speed variations. The maximum torque with diesel fuel operation was 20,15 Nm at 1800 rpm. POSEE's maximum torque was 18,5 Nm at 2100 rpm and POSME's maximum torque was 18,775 Nm at 1500 rpm. The average torque decrease for POSME was 12,5% and for POSEE was 12,36%.

The variations of engine power values in relation with the engine speeds are shown in Fig. 3. The maximum power with diesel fuel operation was 4,93 kW at 2400 rpm. Observed maximum power values of biodiesel fuel operations were also around 2400 rpm but less than the diesel fuel value for each fuel. The average power differences between the diesel fuel and POSME and POSEE were about 16% and 16,9%, respectively.

The specific fuel consumption is one of the important performance parameters of an engine and is defined as the consumption per unit of power in a unit of time. Test results showed that the obtained minimum specific fuel consumption values were in the vicinity of the maximum torque area. As shown in Fig. 4, the minimum specific fuel consumption values were 309,75 gr/kWh with diesel fuel, 322,4 gr/kWh with POSME and 332,4 gr/kWh with POSEE.

The performance values of biodiesels have disadvantages when compared with diesel fuel. These results may be due to the higher viscosity and lower heating values of biodiesels. Because of the higher viscosity, the spraying quality decreases in injectors. So this factor extends the time of fuel evaporating and burning. The fuel burning occurs at expansion time of cylinder and performance decreases<sup>1,15</sup>. The higher exhaust temperatures of biodiesels can indicate that the fuel's energy isn't

used at work time of motor exactly and exits with the exhaust gas.

From the performance viewpoint of biodiesels, POSME has advantages at lower engine speeds and POSEE has advantages at higher engine speeds. POSEE's lower heating value is a little higher than POSME's lower heating value. At that time, the advantage of POSME at lower engine speeds may be attributed to the higher fuel mass flow of the denser and more viscous structure of it. The denser structure of POSME can result in larger mass flow for the same fuel volume, and more viscous fuel means less internal leakage in the fuel pump.

The trends depicted in Fig. 5 for the thermal efficiency indicate that the combustion process proceeded more efficiently with the biodiesels compared to diesel fuel. It can be established that biodiesels possess higher cetane number but we don't know our experiment fuel's cetane number. It is seen here that the effect of higher thermal efficiencies of biodiesels were more than offset by the operation at lower fuel air ratios than the diesel fuel.

### Exhaust emission

In this section, the emissions of carbonmonoxide, unburned hydrocarbons, carbon dioxide, and oxygen, in addition to nitrogen oxide are examined and the results are shown in Figs.6-10 for all the test fuels.

Using of biodiesel fuels resulted in the reduction of unburned HCs and carbonmonoxide (CO) emissions. Fig. 6 illustrates that with average calculation, methyl ester 62%, ethyl ester 43,9% reduced the CO emissions emitted in all cases. Fig.7 exhibit reductions in HC emissions, due to the using POSME and POSEE.

Because of the lower fuel-air equivalence ratios of the biodiesels, poor mixture occurs in the engine and this situation may result with lower HC and CO emissions. Furthermore, since the lower cetane number of the diesel fuel may result in lower tendency to form ignitable mixture, it causes carbonmonoxide and unburned hydrocarbons, which are the products of incomplete combustion<sup>5</sup>.

The NO and NO<sub>x</sub> emissions were increased in all cases when the two biodiesels were used, as shown in Fig. 8 and Fig. 9. These emissions are usually a result of higher combustion temperatures. At the same time, the oxygen concentration is important at these high temperatures. When we look at the fuel-air equivalence ratios, we see the poor mixtures of biodiesels. And also, the chemical structure of esters may increase the oxygen concentration, which is the reactant of nitrogen oxides reaction. The higher NO<sub>x</sub> emissions can be a result of higher exhaust gas temperatures. To reduce NO<sub>x</sub> emissions, the injection timing can be retarded by 2-3 degrees. Also, according to a study<sup>18</sup> reducing the iodine value of biodiesel through enriching the

parent oil in fatty acids with lower iodine value and addition of cetane improvers can eliminate the NO<sub>x</sub> effect.

Fig. 10 illustrate the CO<sub>2</sub> emissions in relation with the engine speeds, respectively. Although POSME's CO<sub>2</sub> emissions are generally lower than the diesel fuel, POSEE's are higher. To explain the reasons of CO<sub>2</sub> emissions, we must know detailed chemical structures of esters and burning reactions.

A study about the exhaust emissions of biodiesels indicates reduction of CO and HC emissions and increase in NO<sub>x</sub> emissions in<sup>19</sup>, too.

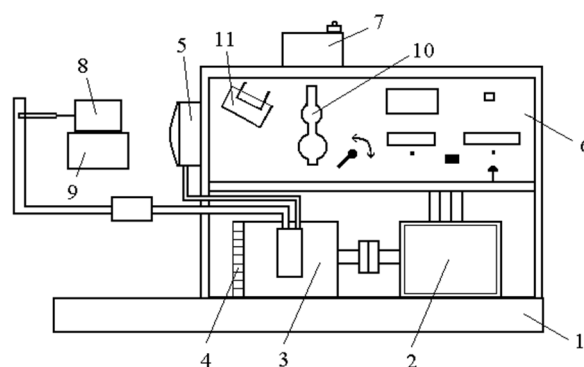
### 6. Conclusions

Compared to diesel fuel, a little amount of power loss and higher specific fuel consumption occurred at biodiesel fuel operations. However, thermal efficiencies of biodiesels were higher. HC and CO emissions of biodiesels were lower than that of diesel fuel, but on the other hand, NO<sub>x</sub> and NO emissions were higher. And also, methyl ester's CO<sub>2</sub> emissions were less.

When we compared POSME and POSEE; performance values are generally same, but the thermal efficiency of POSME is higher. POSME has advantages about CO, HC, CO<sub>2</sub> emissions, too. The price of the methyl alcohol and the succession on transesterification reaction are the other advantages of methyl ester. Because of all these factors, methyl ester is produced and used more than ethyl ester all over the world. But if a country's agriculture is highly developed, ethyl ester can become important. When we look at fuel properties, the lower viscosity and cloud point are the advantages of POSEE according to POSME.

The results of this study show that the biodiesels can be used as an alternative fuel in diesel engines.

### 7. Figure and table



**Fig. 1:** Schematic layout of the test system (1- engine chassis, 2- electrical dynamometer, 3- diesel engine, 4- engine cooling unit, 5- air tank, 6- control unit, 7- fuel tank, 8- exhaust gas analyzer, 9- digital read out of gas analyzer, 10- fuel measurement burette, 11- manometer )

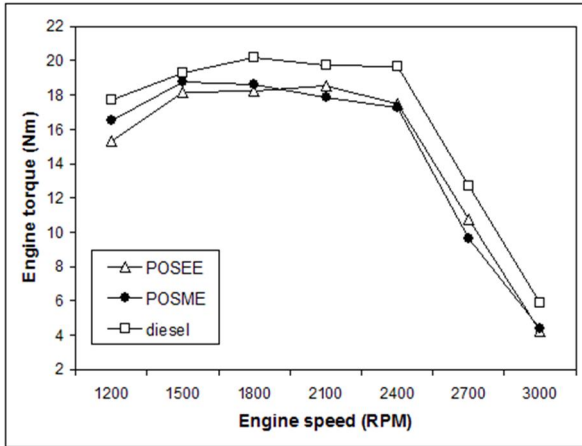


Fig. 2: Engine torque output (Nm) vs. engine rpm for the test fuels.

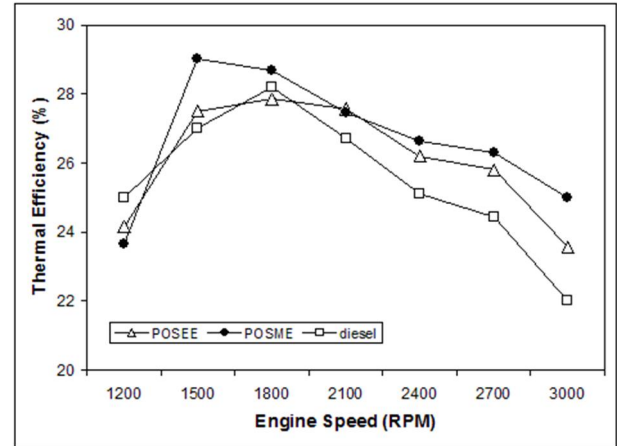


Fig. 5: Thermal efficiency (%) vs. engine rpm for the test fuels.

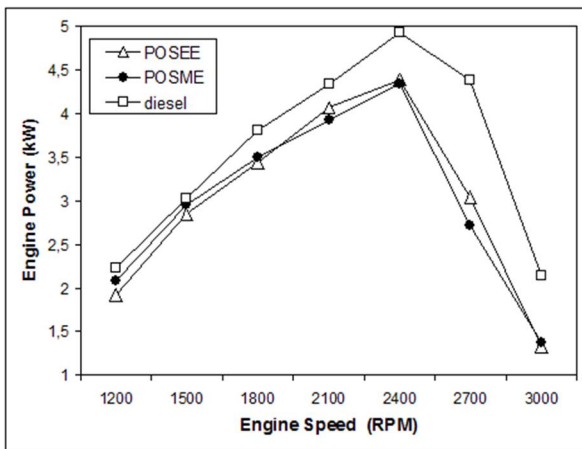


Fig. 3: Engine Power output (kW) vs. engine rpm for the test fuels

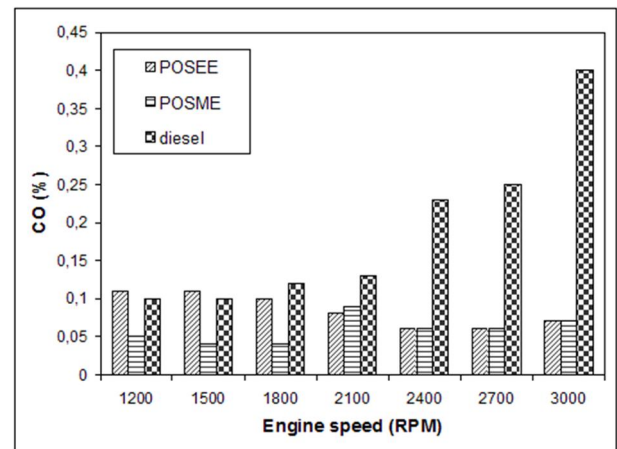


Fig. 6: CO emission (%) vs. engine rpm for the test fuels.

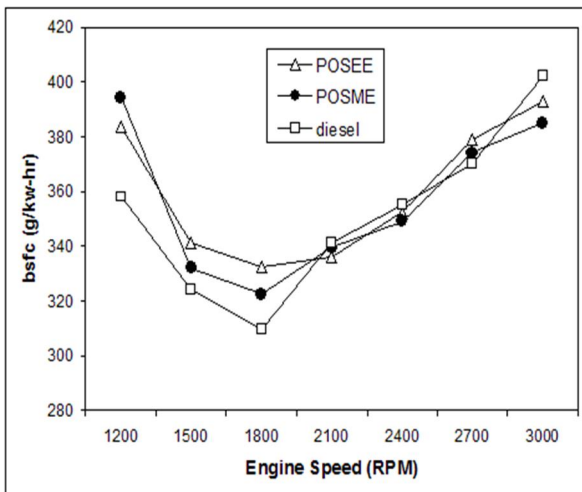


Fig. 4: Brake specific fuel consumption vs. engine rpm for the test fuels.

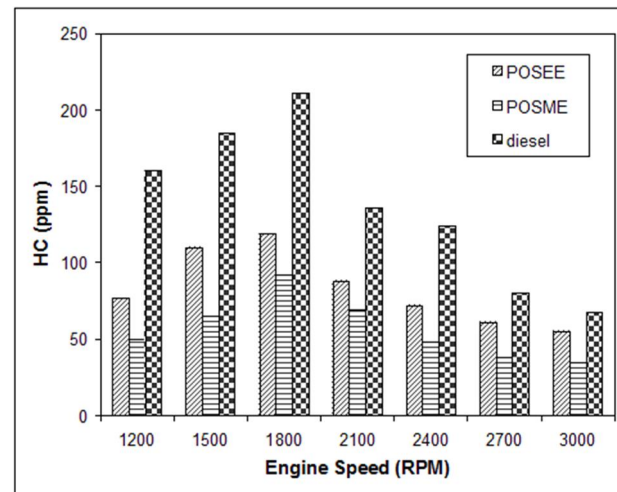


Fig. 7: HC emission (ppm) vs. engine rpm for the test fuels.

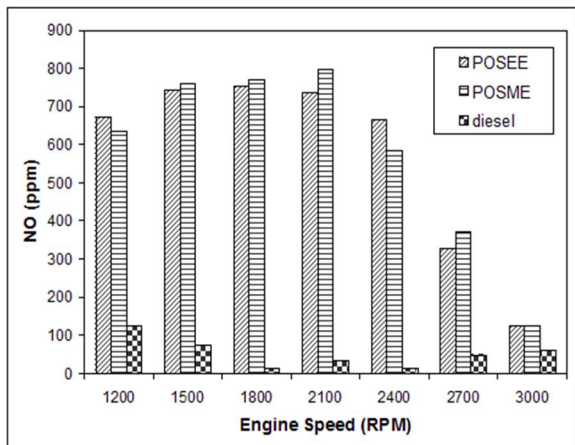


Fig. 8: NO emission (ppm) vs. engine rpm for the test fuels.

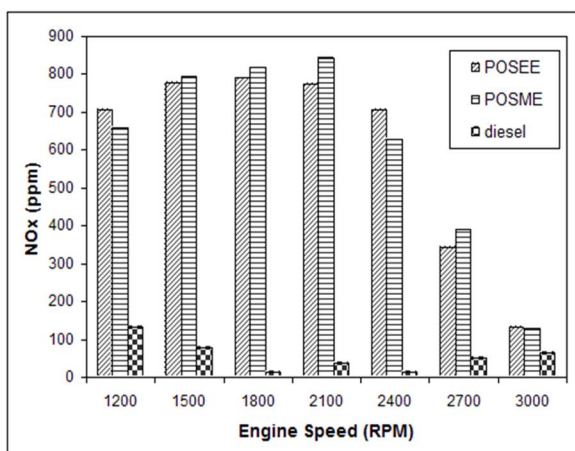


Fig. 9: NO<sub>x</sub> emission (ppm) vs. engine rpm for the test fuels.

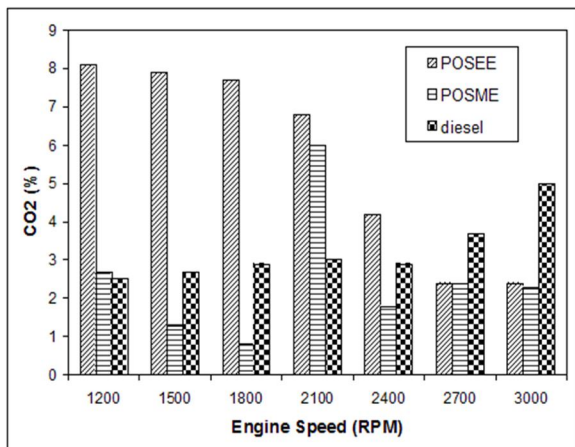


Fig. 10: CO<sub>2</sub> emission (%) vs. engine rpm for the test fuels.

Table 1: Properties of biodiesel products and diesel fuel

Items	POSME	POSEE	Diesel
Appearance	Yellow	Yellow	-----
Specific density (g/cm <sup>3</sup> at 15°C)	0,8866	0,879	0,84
Kinematic viscosity (mm <sup>2</sup> /s at 40°C)	4,97	4,92	3,6
Cloud point (°C)	3	0	-15
Flash point (°C)	160,6	150,6	52
High cal. value (kj/kg)	39781	40128	44631
Lower cal. value (kj/kg)	38631	38882	42700

Table 2: Technical specifications of the diesel test engine

Engine	Lombardini
Model	6LD 400
Type	Air cooled, four strokes
Combustion	Direct injection
Number of cylinders	1
Cylinder diameter	86mm
Stroke	68mm
Compression ration	18:1
Maximum torque	20 Nm at 2200 rpm
Maximum power	6.25 kW at 3600 rpm

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