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# The Pyrolytic Fuel Production From Nutshell-Rice Husk Blends and Determination of Engine Performance and Exhaust Emissions in a Direct Injection Diesel Engine



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### ABSTRACT

In this study, pyrolytic fuel from nutshell and rice husk blends was produced at 500°C temperature, 1 L/min gas flow rate and 10 °C/min heating rate. Pyrolytic fuel was experimented in a single cylinder, direct injection diesel engine with different engine speeds of 1750, 2000, 2250, 2750, 3000 rpm at wide open throttle. The blend of 10% pyrolytic fuel and 90% diesel (B10) and pure diesel were selected as test fuel. Test results showed that conversion efficiency increased with 15% rice husk and 85% nutshell blends compared to pure nutshell at 500°C temperature, 10°C/min and 1L/min gas flow speed. In addition, power output and brake torque decreased 10.20% with B10 compared to diesel. Specific fuel consumption (SFC) increased 28.42% with B10 compared to diesel. HC and CO reduced with pyrolytic fuel about 43.87% and 15.72% compared to diesel respectively. As a result, similar properties were seen between pyrolytic fuel and diesel. So, it was found that pyrolytic fuel could be efficiently used in diesel engines without detailed modification. *Keywords:* Biodiesel, Optimization, Nutshell, Engine Performance, Exhaust Emission

### **1. Introduction**

Fossil fuels have been mostly used in spark ignition and compression ignition engines nowadays. Fossil fuels are generally natural gases or petroleum products. Gasoline and diesel fuels are utilized in transportation sector [1].

Recently, researchers and manufacturers have directed to find alternative energy sources due to increasing of oil prices and imbalance of global oil market. In addition, fossil fuels have undesirable effects on land, air and water. Climate change can negatively affect people [2]. Combustion of fossil fuels caused to release various greenhouse gases such as carbon dioxide, nitrogen oxides and other toxic and volatile compounds. It was determined that carbon dioxide increased by about 10.65 billion tones due to usage of fossil fuels [3]. Researches on alternative energy have increased owing to consumed fossil fuel sources, environmental concerns and political addiction. UN climate panel aimed to reduce greenhouse gases by about 50-80% until 2050. It is essential to get rid of fossil fuels addiction in order to achieve this

goal [4]. The usage of alternative and renewable energy resources (wind, geothermal, hydroelectric, biomass) have become the main topic of conversation due to mentioned issues. Among these resources, biomass has been considered as alternative energy resource and it is important as raw material. It helps to reduce CO<sub>2</sub> emissions [5]. Biomass is one of the available, alternative energy resource that meets huge energy demand for both rural and urban areas. Bioenergy meets 10% of global energy demand and it is known as the biggest renewable energy resource [6]. International energy agency (IEA) has predicted that bioenergy significantly contributes to low carbon global energy systems in road transport, aviation, transportation and industry sectors in the future [7].

Biofuels are liquid or gas that is produced from biomass in order to obtain energy. Biofuels have been considered as environmentally friendly fuel since they have low carbon content and emit low SO<sub>2</sub>, NO<sub>x</sub> and soot emissions [8]. The percentage of the usage of biofuels in transportation is by about 2%. Furthermore, it is aimed to reach 9% with the developed new technologies and methods until 2030. It also contributes to the energy security and on socialeconomic improvement apart from the reduction of greenhouse gases [9]. Gasification, combustion and pyrolysis are the commonly used methods for converting biomass into the of energy forms. Pyrolysis is a thermochemical process that is relatively performed at high temperatures in an oxygen-free environment with low humidity. Raw materials for pyrolysis should be ground into fine pieces in order to obtain higher thermal efficiency [10]. Pyrolysis that is one of the biomass conversion methods, is preferred since it is easily applied and production can be carried out at desired conditions. Pyrolytic fuels can be easily carried and stored. It has been determined that pyrolytic fuels release lower greenhouse gases and emissions compared to fossil fuels [11]. Pyrolysis process is performed in the presence of inert gas (N<sub>2</sub> or Ar) and in the absence of oxygen that reduces environmental pollution because of low gas emissions. This process is convenient for re-use of industrial and agricultural wastes. Efficiencies of obtained products are dependent on the pyrolysis process conditions such as biomass type, particle size,

processing temperature, inert gas flow rate for baring [12].

Nutshell has been utilized as valuable and high calorie fuel (4100–4400 cal/gr) in our country especially in the regions where hazelnut is produced. In our country, about 600,000 tons' hazelnut production has been performed in a year. If it is considered that 50% percentage of produced hazelnut is nutshell, about 300.000 tons' nutshell is obtained in a year [13].

In this study, nutshell was used in order to produce pyrolytic fuel with pyrolysis method. There is no enough study related to pyrolytic fuel production and determination engine performance. There is a knowledge gap in this research field. The main objective of the current study is to increase fuel conversion efficiency in the biodiesel production and observe the engine performance and exhaust emissions. Hence, the effects of temperature on fuel efficiency at constant gas flow and heating rates were experimentally investigated. Rice husk was also added into the nutshell at specific rates (5%, 10%, 15%, 20%) and the effects on conversion efficiency were researched. The variations of brake torque, power output, SFC, CO, HC, NO soot emissions were experimentally and investigated in a diesel engine fueled with the blends of 10% pyrolytic fuel and 90% pure diesel by vol.

# Material and Method Pyrolysis process and fuel properties

Pyrolysis is thermal degradation process in the absence of oxygen [14]. Biomass is decomposed thermally in the atmosphere where the oxygen is absent in the pyrolysis process. The pyrolysis is a highly complex process involving very different reactions. Volatile biomolecules of biomass are decomposed due to heating of biomass and pyrolytic fuel is obtained after condensation. Pyrolysis process is schematically shown in Figure 1.

Nutshell used in the experiments was supplied from a commercial firm. Nutshells are divided into small pieces for the usage in experiments. Pyrolysis device can be able to run between 100 and 550 °C temperature with the heating speed range of 1-10 °C/min. It has also reservoir that can involve 1000 g sample. Pyrolysis device is seen in Figure 2.

The experiments were conducted at 500°C

temperature, 1 L/min gas flow rate and 10 °C/min heating rate. Fuels were produced from 85% hazelnut and 15% rice husk. Table 1 shows the properties of produced fuels.



Figure 1. Schematically view of pyrolysis process



Figure 2. Pyrolysis device

Table 1. Comparison of produced pyrolytic fuel with diesel fuel

Properties	B10	Diesel
Density (g/cm <sup>3)</sup>	0.84168	0.83033
Sulphur (ppm)	90.5	7
Water (ppm)	248.25	35.44
Copper strip	1a	1a
Viscosity (mm <sup>2</sup> /s)	-	2.7



Figure 3. Schematically view of the experimental setup

### 2.2 Engine test bed

Engine performance and exhaust emissions values were determined in the experiments. The test engine was run at wide open throttle and different engine speeds of 1750, 2000, 2250,

2750, 3000 rpm. DC dynamometer was used in order to load test engine. Brake torques, fuel consumption, exhaust emissions were measured and recorded. The schematically view of the experimental setup is shown in Figure 3.

Cussons P8160 DC dynamometer was used in the experiments. Dynamometer can absorb 10 kW engine power at 4000 rpm. Dynamometer can also be operated as motor. Engine load was being varied with strain-gauge load cell placed on dynamometer. Engine speed also can be changed with a potentiometer mounted on dynamometer control panel. A single cylinder, four stroke, air cooled, normally aspirated, direct injection Antor 6LD400 diesel engine was used in the experiments. Technical specifications of the test engine are seen in Table 2.

Table 2. Technical specifications of the test engine		
Model	Antor 6LD400	
Engine type	DI, Diesel	
Cylinder number	1	
Bore (mm)	86	
Stroke	68	
Swept volume (m <sup>3</sup> )	395	
Compression ratio	18:1	
Maximum engine speed	3600	
Power output (kW)	5.4 @3000 rpm	
Brake torque (Nm)	19.6 @2200 rpm	
Number of injector and holes	4x0.24	
Exhaust valve opening timing	21°CA BBDC	
Exhaust valve closing timing	3° CA ATDC	

Table 3. The technical specifications of Bosch BEA 350 model emission analyzer

model emission analyzer		
Measurement	Operating range	Accuracy
CO <sub>2</sub> , % v/v	0 – 18	0.01
CO, % v/v	0 - 10	0.001
HC, ppm	0 – 9999	1 ppm
NO, ppm	0 - 5000	1 ppm
Soot, %	0 - 100	0.1

Table 4. The technical properties of AVL DiSmoke 4000

opacificter		
Analyzer	AVL DiSmoke 4000	
Measurement method	Partial flow	
	Opacity	K value
Operating range	0-100 %	Accuracy 0.1 %
Accuracy [m <sup>-1</sup> ]	0-99.99	0.01

Bosch BEA 350 model emission analyzer was used in order to measure CO, CO<sub>2</sub>, HC, NO emissions. The technical specifications of exhaust gas analyzer are seen in Table 3. The technical properties of AVL DiSmoke 4000 opacimeter are seen in Table 4.

## 3. Results and Discussion

The variations of brake torque versus engine speed are seen in Figure 4. Maximum brake torque was measured at 2200 rpm with the usage of diesel and B10. Higher brake torque was obtained with diesel by about 8.86% compared to B10. Lower brake torque was obtained with B10 due to calorific value. At 2200 rpm, the most charge mixture is delivered into the cylinder in a cycle. So, the highest brake torque was obtained at 2200 rpm. After 2200 rpm, gas leakages and heat losses increase with the increase of engine speed. Hence brake torque decreased. In addition, higher density of B10 caused to poor injection characteristics resulting in worse homogeneity of charge mixture. This situation deteriorated oxidation reactions.



Figure 4. The variations of brake torque with diesel and B10 test fuels



Figure 5. The variations of power output with diesel and B10 test fuels

Figure 5 shows the variations of power output versus engine speed. Maximum power output was obtained at 3000 rpm for both test fuels. Power output was determined as 4.59 kW and 4.22 kW with diesel and B10 respectively. Heat

energy decreases with B10 owing to lower calorific value resulting in lower power output. It was found that brake torque and power output decreased with the usage of pyrolytic fuel. It can be explained that calorific value of pyrolytic diesel. fuels is lower than So, lower performance values were obtained with pyrolytic fuels. Ignition delay increases with pyrolytic fuels. Hence power output and brake torque decrease [15]. The variations of SFC are seen in Figure 6. SFC decreased until a certain value and then started to increase with the increase of engine speed. Minimum SFC was determined at 2500 rpm for all test fuels. SFC increased about 27.64% with B10 compared to diesel at this engine speed. It can be also implied that higher fuel should be ignited in order to obtain same power with B10 compared to diesel. It causes to increase SFC with B10. Higher density of pyrolytic fuel caused to inject more fuel, because fuel particle size is getting bigger with B10 compared to diesel. So, injected fuel increases by mass with B10.



Figure 6. The variations of SFC with diesel and B10 test fuels

It can be mentioned that there is an inverse relationship between calorific value and SFC in the internal combustion engines. The calorific value of test fuel should be higher enough for lower SFC. Lower SFC and higher thermal efficiency are obtained with the increase engine speed, because gas leakages and heat losses decrease towards to partial engine load. Lower mechanical losses are observed between 2500 and 2750 rpm engine speed [16]. The water and oxygen additive content of pyrolytic fuel decreased the calorific value of pyrolytic fuel. This phenomena resulted in higher SFC and the same brake torque with fuel blends compared to pure diesel fuel. Moreover, higher density and viscosity of pyrolytic fuel caused to poor injection characteristics, incomplete combustion and higher fuel consumption [17]. Figure 7 depicts the variations of soot emissions versus engine speed. Soot emission increases at higher engine speed. Minimum soot emissions were measured at 1750 rpm for both test fuels. Soot emission reduced 21.91% with B10 compared to diesel at 1750 rpm.



Figure 7. The variations of soot emissions with diesel and B10 test fuels



figure 8. The effects of B10 and diesel fuels on N formation versus engine speed

Soot emission is released due to poor oxidation of fuel molecules. When the fuel is injected into the combustion chamber. H<sub>2</sub> molecules involving in chemical structure of fuel could be rapidly reacted with O<sub>2</sub> molecules in the cylinder. If unburned HC molecules remaining in the combustion chamber could not meet with O2 molecules, soot particle is formed. Oxidation rate should be increased by the existence of oxygen molecules in combustion regions in the cylinder that is the main reason of the reduction of soot emission. Furthermore, lower soot emissions are obtained with pyrolytic fuel due to lower content of aromatic compounds [18]. Figure 8 shows the effects of B10 and diesel

fuels on NO formation versus engine speed. Minimum NO formation was observed at 1750 rpm. NO increased 7.78% with B10 compared to diesel at this engine speed. NO emissions are released at higher combustion temperature during combustion. Oxygen and nitrogen molecules are reacted at high combustion temperature resulting in NO formation.

Nitrogen oxide reacts with ozone in the atmosphere via ultraviolet light from the sun and nitrogen dioxide is produced. Among these harmful gases, nitrogen oxide is one of the most pollutant gas in our lifecycle. Hydrocarbon compounds are decomposed and they are released as NO and NO<sub>x</sub>. 90% of NO is formed dependent on hydrocarbon compounds. Temperature and oxygen fraction are two main factors that affect the NO formation. Combustion temperature decreases and NO emissions increase with the usage of pyrolytic fuels due to higher water content involving in pyrolytic fuels [19]. The variations of HC emissions versus engine speed are seen in Figure 9. HC increased until 2500 rpm engine speed for both test fuels and then started to decrease after this engine speed. HC reduced 6.93% with B10 compared to diesel at 2500 rpm.



Figure 9. The variations of HC emissions with diesel and B10 test fuels

HC is produced because of incomplete combustion due to low temperature and insufficient oxygen. It was seen that HC increased until 2500 rpm and then started to decrease. Lower HC was measured with pyrolytic fuels at low engine speeds owing to homogeneity and higher oxygen content of pyrolytic fuels. HC increased in the combustion chamber, because higher fuel molecules are delivered into the cylinder at higher engine speeds [20]. On the other hand, injected fuel can be properly diffused in the combustion chamber and gaseous hydrocarbons could not be ignited on the cooler cylinder wall. So, HC increased. However, HC reduced with pyrolytic fuels, because combustion region was small and narrow [21]. The variations of CO emissions with diesel and B10 are seen in Figure 10. It was found that maximum CO was measured at low engine speeds in the experiments. But, CO reduced with the increase of engine speed. CO reduced 15.72% with B10 compared to diesel. Minimum CO was measured at 3000 rpm for both fuels.



Figure 10. The variations of CO emissions with diesel and B10 test fuels

CO can be defined as loss chemical energy that could not be utilized in the engine. The main reason of CO emission release is low air-fuel ratio. CO is formed to CO<sub>2</sub> in case of complete combustion. Combustion could not be completed well due to insufficient oxygen in mixture and low air/fuel exhaust gas temperature. Thus, CO is formed [22]. CO is generally reduced at high engine speeds and loads. At low engine speeds, CO is converted to the CO<sub>2</sub>. Oxidized carbons caused to release higher amount of CO<sub>2</sub> [23].

Combustion efficiency is improved with pyrolytic fuels compared diesel, because they have higher oxygen content. Fuel molecules need sufficient oxygen for complete combustion in the combustion chamber. Since oxygen molecules are abundant during combustion, all fuel molecules can easily participate combustion reactions. Combustion efficiency is dependent on combustion temperature. So, combustion temperature increases at the end of combustion due to sufficient oxygen that chemical oxidation reactions improve. Similarly, combustion efficiency improves. CO reduced because of smaller combustion regions [24]. Another reason is that second atomization is observed via vaporizing water droplet involving in pyrolytic fuel molecules. So, fuel particle can be properly mixed with air. Consequently, lower CO is formed [25]. Figure 11 shows the variations of CO<sub>2</sub> with diesel and B10 test fuels. Minimum CO<sub>2</sub> was measured at 1750 rpm for both fuels. CO<sub>2</sub> increased 23.05% with B10 compared to diesel. CO2 is released due to complete combustion.  $CO_2$  is colorless, odour-free and harmless gas. Since, carbon and hydrogen are disintegrated at the end of combustion, hydrogen is converted to water. If carbon compound meets with sufficient oxygen, it is converted to CO<sub>2</sub> [26]. CO<sub>2</sub> is complete combustion product that carbon molecules could be well oxidized. There is linear relationship between B10 and CO<sub>2</sub> increase. Because, carbon ratio of pyrolytic fuel (C: H = 10.34) is higher than diesel (C: H = 6.47). More carbon molecules were oxidized and higher CO2 was released at the end of combustion [27].



Figure 11. The variations of CO<sub>2</sub> emissions with diesel and B10 test fuels

### 4. Conclusions

In this study, pyrolytic fuel production was performed from nutshells with pyrolysis method. The effects of temperature and rice husk addition on efficiency were investigated and the physical properties of produced pyrolytic fuel were determined. The effects of diesel and B10 test fuels were also investigated on engine performance and exhaust emissions in a single cylinder, direct injection diesel engine. Conversion efficiency was determined as 11.74% in pyrolysis process of pure nutshell at

500°C temperature, 10°C/min and 1L/min gas flow speed. On the other hand, conversion efficiency was determined as 21.73% in pyrolysis process of 15% rice husk and 85% nutshell blends at the same conditions. Fuel efficiency increased by about 85.09% with the addition of 15% rice husk. It is considered that the increase of fuel efficiency is dependent on silicium compound involving in rice husk. Since fuel injectors are clogged with the usage of B10 fuel blends, SPAN80 was added in the experiments. More homogeneous charge mixture was obtained. Experiments were conducted after SPAN 80 was added.

The variations of engine performance and exhaust emissions can be drawn as following.

- Power output and brake torque decreased about 10.20% with B10 compared to diesel.
- SFC increased 28.42% with B10 compared to diesel.
- HC reduced 43.87% with B10 compared to diesel.
- CO reduced by about 15.72% with the usage of pyrolytic fuel compared to diesel.
- It was observed that CO<sub>2</sub> increased 23.05% with pyrolytic fuel compared to diesel.

Researches related to alternative fuels have focused on increase of engine performance, decreasing fuel consumption and exhaust emissions with environmentally friendly fuels. As a result, it was found that the characteristics of pyrolytic fuel obtained from nutshell and rice husk blends are similar to diesel.

### Nomenclature

BBDCBefore bottom dead centerCOCarbon monoxideCO2Carbon dioxideDIDirect injectionHCHydrocarbonIEAInternational Energy AgencyNOxNitrogen oxidesSFCSpecific fuel consumptionSO2Sulphur dioxideUNUnited nation	ATDC	After top dead center
COCarbon monoxideCO2Carbon dioxideDIDirect injectionHCHydrocarbonIEAInternational Energy AgencyNOxNitrogen oxidesSFCSpecific fuel consumptionSO2Sulphur dioxideUNUnited nation	BBDC	Before bottom dead center
CO2Carbon dioxideDIDirect injectionHCHydrocarbonIEAInternational Energy AgencyNOxNitrogen oxidesSFCSpecific fuel consumptionSO2Sulphur dioxideUNUnited nation	CO	Carbon monoxide
DIDirect injectionHCHydrocarbonIEAInternational Energy AgencyNOxNitrogen oxidesSFCSpecific fuel consumptionSO2Sulphur dioxideUNUnited nation	$CO_2$	Carbon dioxide
HCHydrocarbonIEAInternational Energy AgencyNOxNitrogen oxidesSFCSpecific fuel consumptionSO2Sulphur dioxideUNUnited nation	DI	Direct injection
IEAInternational Energy AgencyNOxNitrogen oxidesSFCSpecific fuel consumptionSO2Sulphur dioxideUNUnited nation	HC	Hydrocarbon
NOxNitrogen oxidesSFCSpecific fuel consumptionSO2Sulphur dioxideUNUnited nation	IEA	International Energy Agency
SFCSpecific fuel consumptionSO2Sulphur dioxideUNUnited nation	NO <sub>x</sub>	Nitrogen oxides
SO2Sulphur dioxideUNUnited nation	SFC	Specific fuel consumption
UN United nation	$SO_2$	Sulphur dioxide
	UN	United nation

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