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Semi-Analytical and Finite Element Investigations of the Vibration of a Stepped Beam on an Elastic Foundation

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Abstract

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In this study free vibration behavior of a stepped beam on an elastic foundation is considered. The vibration of uniform beams on an elastic foundation has been previously studied extensively and various solutions are available in the literature. However, the problem considered in current study appears not to have been widely covered in the literature and analytical solutions are strictly limited. To this aim, semi-analytical solutions are obtained first by using Adomian decomposition method, then finite element solutions are computed via structural finite element analysis software (SAP 2000). The free vibration analysis of stepped beam considering the combinations of different support conditions at each end are performed employing semi-analytical and finite element methods. The findings of the analysis are compared and discussed in detail.

Keywords: segmented beam, elastic foundation, vibration, Adomian decomposition method

1. INTRODUCTION

Beam on elastic foundation problems are of great interest for researchers in the fields of civil, mechanical and aeronautical engineering related to the design of structural members of buildings, aircrafts, pipes, railroads, etc.

In the literature, numerous studies were conducted on the vibration analysis of beams [1- 3]. Researchers also focused on the special cases such as stepped beams [4] and beams on elastic foundations [5]. Wang [6], Kukla [7] and Belles et al. [8] are three interesting contributions to the technical studies about the subject. Thambiratnam and Zhuge [9] developed a simple finite element method and applied to treat the free vibration of

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analysis of Euler-Bernoulli beams on elastic foundations for different cases. Durgun [10] also conducted a study on the analysis of stepped beams on elastic foundation by employing Homotopy Perturbation Method (HPM). Durgun [8] solved the same numerical example with Wang [6] and obtained exactly the same results up to four decimals. The solution proposed by Wang [6] was tedious and is only valid for specific cases and he reported the results only for the beams having free-free and simply supported end conditions. In addition to Wang's work, Durgun [10] also presented the results for fixed-fixed and fixed-hinged end condition cases. However, Durgun [10] has not verified his findings regarding the fixed-fixed and fixed-hinged end conditions with other reliable methods in the literature.

The main aim of this study is to provide semianalytical and finite element solutions to the same problem in order to verify the results obtained by using HPM [10] and fulfill the lack of analytical solutions provided in the work by Wang [6].

2. VIBRATION OF STEPPED BEAM ON ELASTIC FOUNDATION

In this section, vibration of an Euler beam on Winkler foundation is considered that are the most widely used models for the beams on elastic foundations. Such a beam is shown in Figure 1.

Figure 1 Stepped beam on elastic foundation

Equation of motion for a uniform Euler beam on Winkler foundation is given as:

$$
EI\frac{\partial^4 y(x,t)}{\partial x^4} + \rho A \frac{\partial^2 y(x,t)}{\partial t^2} + k y(x,t) = 0 \qquad (1)
$$

where *EI* is the flexural rigidity of the beam, ρ is the density per unit volume, A is the crosssectional area, k is the stiffness of Winkler foundation and y is the transverse displacement.

In order to find the eigenfrequencies ω of the beam one may assume

$$
y(x,t) = w(x) e^{i\omega t}
$$
 (2)

Substituting Eq.(2) in Eq.(1) yields

$$
EI\frac{d^4w(x)}{dx^4} - \rho A\omega^2 w(x) + kw(x) = 0 \tag{3}
$$

Eq.(3) can be rearranged as

$$
\frac{d^4w(x)}{dx^4} - \left(\lambda^4 - \frac{k}{EI}\right)w(x) = 0\tag{4}
$$

where $\lambda^4 = \omega^2 \rho A / EI$. Introducing the parameter β , such that, $\beta^4 = \lambda^4 - \frac{k}{E}$ $\frac{\kappa}{EI}$ solution to Eq.(4) is

$$
w(x) = C_1 \sin(\beta x) + C_2 \cos(\beta x)
$$

$$
+ C_3 \sinh(\beta x) + C_4 \cosh(\beta x) \quad (5)
$$

when $\beta^4 \ge 0$ which describes that foundation stiffnes has no effect in the solution. However, if the vibration frequency of the beam is relatively low, *i.e.*, β^4 < 0 the solution of Eq.(4) becomes

$$
w(x) = C_1 \sin\left(\frac{\beta x}{\sqrt{2}}\right) \sinh\left(\frac{\beta x}{\sqrt{2}}\right) + C_2 \sin\left(\frac{\beta x}{\sqrt{2}}\right) \cosh\left(\frac{\beta x}{\sqrt{2}}\right) + C_3 \cos\left(\frac{\beta x}{\sqrt{2}}\right) \sinh\left(\frac{\beta x}{\sqrt{2}}\right) + C_4 \cos\left(\frac{\beta x}{\sqrt{2}}\right) \cosh\left(\frac{\beta x}{\sqrt{2}}\right) \tag{6}
$$

which is a different solution when compared to Eq.(5). The coefficients C_1 , C_2 , C_3 , and C_4 in both solutions can be evaluated according to boundary conditions at the supports. These conditions are given as follows:

- For free end $w''(x) = 0$ and $w'''(x) = 0$
- For hinged end $w(x) = 0$ and $w''(x) = 0$
- For clamped end $w(x) = 0$ and $w'(x) = 0$

There are also four boundary conditions due to continuity at the junction of two segments of the stepped beam. These conditions impose the equality of the displacement, the slope, the moment and the shear force at the junction and given as follows:

$$
\bullet \quad w_1(x) = w_2(x)
$$

- $w'_1(x) = w'_2(x)$
- $w_1''(x) = \alpha w_2''(x)$
- $w_1'''(x) = \alpha w_2'''(x)$

where $\alpha = I_2/I_1$.

3. ADOMIAN DECOMPOSITION METHOD

In Adomian decomposition method (ADM) a general form of the following differential equation is assumed.

$$
Lu + Nu + Ru = g(x) \tag{7}
$$

where $u(x)$ is the unknown solution, $g(x)$ is the source term, L is the linear operator, N is the nonlinear operator and R is the operator for remainder terms. The solution to Eq.(7) is

$$
u(x) = f(x) - L^{-1}(Nu) - L^{-1}(Ru)
$$
 (8)

where L^{-1} is the inverse linear operator and $f(x) = L^{-1}(g(x))$. The solution is constructed with an infinite series in the following form

$$
u(x) = \sum_{n=0}^{\infty} u_n(x) \tag{9}
$$

The nonlinear term Nu is represented by so-called Adomian polynomials given below.

$$
Nu = \sum_{n=0}^{\infty} A_n(u_0, u_1, \cdots, u_n)
$$
 (10)

where A_n is the n^{th} Adomian polynomial defined as the following term.

$$
A_n = \frac{1}{n!} \frac{d^n}{d\lambda^n} N(\sum_{n=0}^{\infty} \lambda^k u_k)
$$
 (11)

The method leads to successive approximations as follows:

$$
u_0(x) = f(x) \tag{12}
$$

$$
u_n(x) = -L^{-1}(Ru_{n-1} - A_{n-1})
$$
 (13)

Finally, the solution is calculated by adding the successive approximations given in Eqs.(12-13). An Nth order analytical approximation includes the terms up to u_N as given below

$$
u(x) = \sum_{n=0}^{N} u_n(x) \tag{14}
$$

For further details of the method, the reader may refer to [11].

4. ADM SOLUTION OF THE PROBLEM

An initial approximation based on Eq.(12) may be obtained as

$$
w_0(x) = Ax^3 + Bx^2 + Cx + D \tag{15}
$$

where $A = y'''(0)/6$, $B = y''(0)/2$, $C = y'(0)$ and $D = y(0)$. Successive approximations for an Nth order solution may be computed according to Eq. (13) as

$$
w_n(x) = L^{-1}(\beta^4 y_{n-1}), \quad n > 0 \tag{16}
$$

Since there are two segments in the stepped beam, an initial approximation of the form given in Eq.(15) is assumed for both segments of the beam.

$$
w_0^{(1)}(x) = A_1 x^3 + B_1 x^2 + C_1 x + D_1 \tag{17}
$$

$$
w_0^{(2)}(x) = A_2 x^3 + B_2 x^2 + C_2 x + D_2 \tag{18}
$$

Eight boundary conditions are required to determine eight unknowns introduced in Eqs.(17) and (18). These conditions are four boundary conditions at the supports and four continuity conditions. Hence eight equations in eight unknowns are produced can be represented in the following matrix form.

$$
[K]_{8\times8} \{\Lambda\}_{8\times1} = \{0\}_{8\times1} \tag{19}
$$

where $[K]$ includes the term β which is the function of vibration frequency ω and the unknown vector {Λ} includes unknown coefficients in the initial approximations in Eqs.(17) and (18). The trivial solution of Eq.(19) corresponds the undeformed beam. Hence, a nontrivial solution to the problem can be obtained by equating the determinant of coefficient matrix to zero that lead to free vibration frequencies of the stepped beam on elastic foundation considered.

5. NUMERICAL APPLICATION

Wang [6] calculated analytical solutions for natural frequencies of two stepped beams on elastic foundation, one simply supported and one with free ends. There are no available analytical solutions for the beam with both ends clamped, the beam with one end clamped and one end simply supported and the beam with one end clamped and one end free. For simplicity following abbreviations are used for different combinations of end conditions.

- FF Free Free
- SS Both ends simply supported
- CC Clamped Clamped
- CS Clamped Simply supported

Wang [6] performed the analysis for the following data: $E = 6.50 \times 10^{11}$ Pa, $\rho = 213.60$ kg/m², $H_1 = 0.10$ m, $H_2 = 0.15$ m, $B = 0.08$ m, $L =$ 5.00 m. Two different foundation modulus were used in the calculations, $v = 1/100$ and $v =$ 1/200 where v is defined as k/El . μ denotes the ratio of the length of thinner segment having depth of H1 to total length L.

SAP2000 [12] was employed for obtaining finite element (FE) solutions. Line springs that can be assigned in any of the local axes direction of a frame object is defined in vertical direction to simulate the elastic foundation. SAP2000 [12] distributes the springs associated with the frame object to all of the nodes (Figure 2).

Figure 2 FE model for simply supported stepped beam on elastic foundation (50 beam elements)

Table 1 compares the previous results (analytical [6], HPM [10]) with the results of present study (ADM, FE solution) for natural frequencies of stepped FF beam on elastic foundation. ADM solutions are conducted to $12th$ order and computations are identical to the analytical [6] and HPM [10] solutions. FE solutions are also in excellent agreement for the first two frequency values.

In Table 2 only first two natural frequencies are available for SS beams [6, 10] and used for comparison. However, first three frequencies are computed for stepped SS beam on elastic foundation using ADM and FEM. ADM results are in excellent agreement with previous results while FE solutions are in very good agreement with the same results for which first two frequencies have the same accuracy of analytical solution.

Analytical solutions for the first three frequencies for stepped CC and CS beams on elastic foundation are not available in the literature. Only HPM solutions [10] exist and the results of this study for these cases are compared with only HPM results.

In Table 3 and Table 4 it seems that ADM solutions are in excellent agreement with HPM results [10]. FE solutions are in very good agreement with both results as in previous cases.

Table 1 Natural frequencies of FF stepped beam on elastic foundation

$v = 1/100$												
	Wang $[6]$, Durgun $[10]$			ADM			FE (50 Elements)			FE (100 Elements)		
μ	İ1	f_2	f_3	f_1	f_2	f_3		f ₂	f3		f_2	f3
	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)
Ω	5.8531	5.8531	96.4081	5.8531	5.8531	96.4081	5.8531	5.8531	96.2888	5.8531	5.8531	96.3783
0.1	5.8534	6.2201	101.0194	5.8534	6.2201	101.0194	5.8534	6.2199	100.8885	5.8534	6.2201	100.9867
0.2	5.8557	6.5291	100.7912	5.8557	6.5291	100.7912	5.8558	6.5287	100.6555	5.8557	6.5290	100.7573
0.3	5.8635	6.7692	93.4247	5.8635	6.7692	93.4247	5.8636	6.7688	93.2962	5.8636	6.7691	93.3926
0.4	5.8821	6.9403	82.2168	5.8821	6.9403	82.2168	5.8822	6.9399	82.1076	5.8821	6.9402	82.1895
0.5	5.9194	7.0513	72.8477	5.9194	7.0513	72.8477	5.9196	7.0509	72.7549	5.9195	7.0512	72.8245
0.6	5.9876	7.1161	66.7402	5.9876	7.1161	66.7402	5.9876	7.1161	66.6565	5.9876	7.1160	66.7193
0.7	6.1046	7.1494	63.2525	6.1046	7.1494	63.2525	6.1048	7.1492	63.1733	6.1047	7.1494	63.2327
0.8	6.2993	7.1637	61.5574	6.2993	7.1637	61.5574	6.2996	7.1636	61.4814	6.2994	7.1637	61.5384
0.9	6.6214	7.1680	61.4321	6.6214	7.1680	61.4321	6.6216	7.1679	61.3585	6.6214	7.1680	61.4137
1.0	7.1685	7.1685	64.5528	7.1685	7.1685	64.5528	7.1685	7.1685	64.4735	7.1685	7.1685	64.5330

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Table 2 Natural frequencies of SS stepped beam on elastic foundation

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Table 4

j.

Natural frequencies of CS stepped beam on elastic foundation

$v = 1/100$												
	Durgun $[10]$			ADM			FE (50 Elements)			FE (100 Elements)		
μ	İ1	f_2	f_3	ħ	f ₂	f3	Ť1	f_2	f3		$\mathbf{1}_2$	f3
	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)
Ω	66.5734	214.9847	448.4203	66.5734	214.9847	448.4203	66.5731	214.9836	448.4177	66.5731	214.9837	448.4182
0.1	56.5047	195.8623	420.3049	56.5047	195.8623	420.3049	56.5049	195.8703	420.3450	56.5046	195.8636	420.3136
0.2	56.2305	192.4146	399.6111	56.2305	192.4146	399.6111	56.2318	192.4243	399.5628	56.2307	192.4164	399.5977
0.3	55.8497	181.3330	393.1700	55.8497	181.3330	393.1700	55.8516	181.3177	393.1022	55.8500	181.3286	393.1518
0.4	53.7662	176.7855	386.3098	53.7662	176.7855	386.3098	53.7671	176.7596	386.3394	53.7662	176.7784	386.3160
0.5	50.7342	177.9354	355.5730	50.7342	177.9354	355.5730	50.7338	177.9263	335.5419	50.7338	177.9325	355.5641
0.6	47.9739	172.5648	346.3939	47.9739	172.5648	346.3939	47.9714	172.5730	346.3369	47.9731	172.5662	346.3785
0.7	46.0879	159.7651	342.7486	46.0879	159.7651	342.7486	46.0845	159.7659	342.7674	46.0869	159.7647	342.7522
0.8	45.1264	148.8735	319.4041	45.1264	148.8735	319.4041	45.1230	148.8606	319.4003	45.1254	148.8697	319.4021
0.9	44.8212	144.0672	301.8586	44.8212	144.0672	301.8586	44.8188	144.0499	301.8069	44.8204	144.0624	301.8447
1.0	44.7878	143.4492	299.0073	44.7878	143.4492	299.0073	44.7876	143.4485	299.0056	44.7876	143.4486	299.0059
$v = 1/200$												

Figure 3 Effect of foundation stiffness on natural frequencies of stepped FF beam

Effect of foundation stiffness on first two frequencies is clearly illustrated in Fig.3 However, foundation effect for the third natural frequency is indistinguishable.

For stepped SS, CC and CS beams there are no significant difference in the natural frequencies for $v = 1/100$ and $v = 1/200$. Hence, between Figs. 4 and 6 only the variation of first three frequencies is plotted for $\nu = 1/100$.

Figure 4 Variation of natural frequencies of stepped SS beam

Figure 5 Variation of natural frequencies of stepped CC beam

Figure 6 Variation of natural frequencies of stepped CS beam

It can be mentioned that natural frequencies of the stepped SS, CC and SS beams decreases with increasing μ values considering the variations shown in figures 4-6.

6. CONCLUSIONS

In this study, natural frequencies of stepped beams on elastic foundations are investigated via ADM and FEM. Analytical solutions for this problem are available for the beam with free ends and for simply supported beam. There are no other available analytical solutions for the beam with both ends clamped and for the beam with one end clamped and one end simply supported. All four cases previously were solved using HPM. ADM solutions of this study are in perfect agreement with analytical and HPM solutions. FE solutions are computed employing SAP 2000 software by using 50-element and 100-element models. Both FE models produced reliable results when compared ADM and previously available solutions.

Effect of foundation modulus is found to be distinguishable only for stepped FF beam considering the two different foundation modulus investigated in the scope of study. Variation of frequencies with the position of the intersection point of the two segments is also depicted graphically for stepped SS, CC and CS beams for which it is observed that natural frequencies decreases while μ increases. This result makes sense; As μ value increases, the frequency is

mostly dominated with segment one that is having relatively low moment of inertia.

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The Declaration of Conflict of Interest/ Common Interest

No conflict of interest or common interest has been declared by the authors.

Authors' Contribution

In this study, the contributions of the authors during the research, analysis, submission, review and editing stages are equal.

The Declaration of Ethics Committee Approval

The authors declare that this document does not require an ethics committee approval or any special permission.

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